

$m\ddot{x} = F(t, x, v)$

Eqns of Motion

- If  $F = F(t)$

$m \frac{dv}{dt} = F(t)$

$v(t) = v_0 + \frac{1}{m} \int F(t') dt'$

$x(t) = x_0 + \int_{t_0}^t v(t') dt'$

- If  $F = F(v)$

$m \frac{dv}{dt} = F(v) \Rightarrow \frac{dv}{F(v)} = \frac{dt}{m}$

$\int_{v(t)}^{v(t')} \frac{dv'}{F(v')} = \frac{1}{m} (t - t_0)$

$x(t) = \int v(t) dt$

- If  $F = F(x)$

$m \frac{dv}{dt} = F(x) \quad \frac{dv}{dt} = \frac{dv}{dx} \frac{dx}{dt} = \frac{dv}{dx} v$

$\rightarrow m v \frac{dv}{dx} = F(x)$

$m \int v dv = \int F(x) dx \Rightarrow \frac{1}{2} mv^2 = \int F(x) dx$

Kinetic Energy = Work Done

$F = -\frac{dV}{dx} \rightarrow \frac{1}{2} mv^2 + V(x) = c$

Conservation of Energy

$\int F dt = \Delta p$

Equilibrium:  $F(x_0) = 0$

Let  $x = x_0 + \delta \rightarrow F(x) \approx F(x_0) + (x - x_0) F'(x_0) + \dots$

$\approx (x - x_0) F'(x_0)$

If  $k = -F'(x_0)$

$F(x) \approx -k(x - x_0)$  ← harmonic oscillator!

$\Rightarrow m\ddot{x} = -k(x - x_0)$

$\rightarrow \ddot{x} + \omega^2(x - x_0) = 0$  for  $\omega^2 = \frac{k}{m}$

Stable if  $F'(x_0) < 0 \quad F \approx -k(x - x_0)$

Unstable if  $F'(x_0) > 0 \quad F \approx k(x - x_0)$

$F = -\frac{dV}{dx} \Rightarrow \text{eq @ } \frac{dV}{dx} = 0$

$V(x) \approx V(x_0) + \frac{1}{2} V''(x_0)(x - x_0)^2$

$\rightarrow F = -V''(x_0)(x - x_0)$

$\rightarrow k = -V''(x_0)$

$V'' > 0 \Rightarrow \text{min} \Rightarrow \text{stable}$

$V'' < 0 \Rightarrow \text{max} \Rightarrow \text{unstable}$

General Osc. Soln.

$x(t) = A \cos(\omega_0(t - t_0)) + B \sin(\omega_0(t - t_0))$

Normal Modes

$\vec{x} = \begin{pmatrix} x_1 \\ x_2 \\ \vdots \\ x_n \end{pmatrix} v_i, m_i \ddot{x}_i = F_i(x_1, \dots, x_n)$

$K_{ij} = -\left. \frac{\partial F_i}{\partial x_j} \right|_0 \Rightarrow F_i = -\sum_j K_{ij} x_j$

Then,  $M \ddot{\vec{x}} = -K \vec{x}$  for  $M = \text{diag}(\epsilon m_i^3)$

$\equiv M \ddot{\vec{x}} + K \vec{x} = 0$

Near eq.  $x=0, F_i(\vec{x}) = \sum_j \left. \frac{\partial F_i}{\partial x_j} \right|_0 x_j$

Set  $\vec{x} = \vec{v} \exp(i\omega t)$

$\Rightarrow \ddot{\vec{x}} = -\omega^2 \vec{v} \exp(i\omega t)$

$\rightarrow M(-\omega^2 \vec{v} \exp(i\omega t)) + K \vec{v} \exp(i\omega t) = 0$

$\Rightarrow (-\omega^2 M + K) \vec{v} = 0 \equiv K \vec{v} = \omega^2 M \vec{v}$

triv. val pr. ob.

Nontrivial if  $\det(K - \omega^2 M) = 0$

determinant allowed  $\omega^2$

If  $\vec{x}(t) = \vec{v} \cos(\omega t)$

$\vec{x} = \sum q_n(t) v_n$  where

each  $q_n$  satisfies

$\ddot{q}_n + \omega_n^2 q_n = 0$

Forced Oscillations

$\ddot{x} + \omega_0^2 x = \frac{f(t)}{m}$

General structure:

$x(t) = x_{\text{hom}}(t) + x_{\text{part}}(t)$

$x_{\text{hom}}(t) = A \cos(\omega_0 t) + B \sin(\omega_0 t)$

Let  $f(t) = F_0 \cos(\omega t)$

$x(t) = A \cos(\omega t)$

w/  $A = \frac{F_0}{m(\omega_0^2 - \omega^2)}$

If  $\omega \rightarrow \omega_0, A \rightarrow \infty$ , energy builds up, large osc.

$\omega \ll \omega_0$ , system follows force slowly

$\omega \gg \omega_0$ , system can't keep up.

Arbitrary  $F(t)$ :

Fourier:  $f(t) = \sum f_n \cos(n\omega_0 t)$

L solve each & add

Green's Fn:  $f(t) = \int f(t') S(t-t') dt'$

L works b/c DE is linear

$G + \omega_0 G = \delta(t-t')$

$\rightarrow G(t, t') = \frac{1}{\omega_0} \sin(\omega_0(t-t')) \Theta(t-t')$

$\rightarrow x(t) = \frac{1}{m\omega_0} \int_{-\infty}^t \sin(\omega_0(t-t')) f(t') dt'$

undamped case,  $\Theta(x) = \begin{cases} 1 & \text{if } x > 0 \\ 0 & \text{else} \end{cases}$

Gravitation

for a point mass  $\vec{F} = -\frac{GMm}{r^2} \hat{r}$

$d\vec{F} = -\frac{Gm dm}{r^2} \hat{r}, \vec{F} = \int d\vec{F}$

Thin Spherical Shell:

$F(r) = \begin{cases} -\frac{GMm}{r^2} & r > R \\ 0 & r \leq R \end{cases}$

$V(r) = -\int \frac{Gm dm}{L} \quad F_r = -\frac{dV}{dr}$

center of shell @ origin

$L = \int R^2 + r^2 - 2Rr \cos(\theta)$

$V(r) = -\int \frac{Gm dm}{L} \quad dA = 2\pi R^2 \sin\theta d\theta$

set  $u = L^2; \quad V \propto [(R+r) - |R-r|]$

Solid sphere, uniform density:

$F(r) = \begin{cases} -\frac{GMm}{r^2} & r > R \\ -\frac{GMm}{R^3} r & r < R \end{cases}$

$M(r) = M \frac{r^3}{R^3}$  inside:  $F \propto r$

SHM w/  $\omega^2 = \frac{GM}{R^3}$

Infinite sheet:  $F = -2\pi G \sigma m$

Symmetry  $\rightarrow$  only vertical force

$L = \sqrt{z^2 + x^2} \quad dF = -\frac{Gm dm}{L^2} \frac{x}{L}$

$\int_0^\infty \frac{x dz}{(z^2 + x^2)^{3/2}} = \frac{1}{x}$  (rad R)

sheet w/ circular hole:

$F(x) = -\frac{2\pi G \sigma m x}{\sqrt{R^2 + x^2}}$

$F_{\text{sheet above}} \quad F_{\text{disk}} = -2\pi G \sigma m \left( \frac{x}{\sqrt{x^2 + R^2}} \right)$

$F_{\text{hole}} = F_{\text{sheet}} - F_{\text{disk}}$

Gauss' Law

$\nabla \cdot \vec{g} = -4\pi G \rho$

$\oint \vec{g} \cdot d\vec{A} = -4\pi G M_{\text{enc}}$

$\vec{g} = -\nabla \Phi$

Vector Calc Stuff

$\nabla \cdot \vec{F} = \text{div} \quad \nabla \times \vec{F} = \text{curl}$

$\nabla^2 = \nabla \cdot \nabla \vec{F}, \int_V (\nabla \cdot \vec{F}) dV = \oint_S \vec{F} \cdot d\vec{A}$

$\int_S (\nabla \times \vec{F}) \cdot d\vec{A} = \oint_{\partial S} \vec{F} \cdot d\vec{r}$

$\nabla \times \nabla f = 0 \quad \nabla \cdot (\nabla \times \vec{F}) = 0$

$a \times b = \begin{vmatrix} \hat{i} & \hat{j} & \hat{k} \\ a_x & a_y & a_z \\ b_x & b_y & b_z \end{vmatrix}$

Plane  $ax + by + cz + d = 0$

line  $r_1 + r_2$

Forced Harmonic Motion

$m\ddot{x} + c\dot{x} + kx = F_0 \sin(\omega t)$   
 $x(t) = x_{tr}(t) + x_{steady}(t)$   
 $x(t) = X \sin(\omega t - \phi)$   
 $X(\omega) = \frac{F_0}{\sqrt{(k - m\omega^2)^2 + (c\omega)^2}}$   
 $\tan \phi = \frac{c\omega}{k - m\omega^2}$   
 $\omega \rightarrow 0 \quad X = \frac{F_0}{k}$   
 $\omega \rightarrow \infty \quad X = \frac{F_0}{m\omega^2}$   
 Complex Method:  
 $x(t) = \tilde{A} e^{-i\omega t}$   
 $\tilde{A} = \frac{F_0}{k - m\omega^2 + ic\omega}$   
 $|\tilde{A}| = \frac{F_0}{\sqrt{(k - m\omega^2)^2 + (c\omega)^2}}$

Fourier Decomp:

for periodic  $F(t)$  w/ period  $T$ ,  $\omega_n = \frac{2\pi}{T}$   
 $F(t) = \frac{a_0}{2} + \sum_{n=1}^{\infty} [a_n \cos(n\omega_n t) + b_n \sin(n\omega_n t)]$   
 $a_n = \frac{2}{T} \int_0^T F(t) \cos(n\omega_n t) dt$   
 $b_n = \frac{2}{T} \int_0^T F(t) \sin(n\omega_n t) dt$   
 $a_0 = \frac{2}{T} \int_0^T F(t) dt$   
 or  $a_n \cos(n\omega_n t) + b_n \sin(n\omega_n t) = A_n \sin(n\omega_n t + \phi_n)$  for  
 $A_n = \sqrt{a_n^2 + b_n^2}$   
 if not periodic  
 $F(t) = \int_{-\infty}^{\infty} \tilde{F}(\omega) e^{i\omega t} d\omega$   
 w/  $\tilde{F}(\omega) = \frac{1}{2\pi} \int_{-\infty}^{\infty} F(t) e^{-i\omega t} dt$

Gravitation of a Rod

rod on x axis;  $\lambda$  mass density;  
 test mass  $m$  off rod  
 $dF = \frac{G m dm}{r^2} \quad dg = G \frac{dm}{r^2}$  toward source  
 $dm = \lambda dx$   
 stable if  $v''_{eff}(z_0) > 0$   
 unstable if  $v''_{eff}(z_0) < 0$   
 let  $u = \frac{1}{z}$ :  $\frac{du^2}{d\theta^2} + u = -\frac{m}{l^2 u^2} F(\frac{1}{u})$  for  $F(z) = -\frac{dV}{dz}$

Variations  $L = T - V + \sum \lambda_a f_a$

$S[q] = \int_{t_1}^{t_2} L(q, \dot{q}, t) dt$   
 $\delta S = 0$  for physical path  
 $q(t) \rightarrow q(t) + \epsilon \eta(t) \quad \delta q = \eta$   
 $\delta \dot{q} = \frac{d}{dt} \delta q \quad \delta q(t_1) = \delta q(t_2) = 0$   
 $\delta S = \int_{t_1}^{t_2} \left( \frac{\partial L}{\partial q} \delta q + \frac{\partial L}{\partial \dot{q}} \delta \dot{q} \right) dt$   
 Euler-Lagrange:  $\frac{d}{dt} \left( \frac{\partial L}{\partial \dot{q}_i} \right) - \frac{\partial L}{\partial q_i} = \sum \lambda_a \frac{\partial f_a}{\partial q_i}$   
 boundary term  $\left[ \frac{\partial L}{\partial \dot{q}} \delta q \right]_{t_1}^{t_2}$   
 fixed endpoints = 0, free gives conditions

Hamiltonian Mechanics

$P_i = \frac{\partial L}{\partial \dot{q}_i} \quad H = T + V$   
 $\dot{q}_i = \frac{\partial H}{\partial P_i} \quad \dot{P}_i = -\frac{\partial H}{\partial q_i}$   
 $\frac{dH}{dt} = \frac{\partial H}{\partial t} = 0 \Rightarrow H$  conserved  
 $\frac{\partial H}{\partial q_i} = 0 \Rightarrow q_i$  cyclic  
 $\frac{\partial H}{\partial P_i} = 0 \Rightarrow P_i$  conserved  
 $H(q_i, P_i, t) = \sum P_i \dot{q}_i - L(q_i, \dot{q}_i, t)$   
 $\{f, g\} = \sum_i \left( \frac{\partial f}{\partial q_i} \frac{\partial g}{\partial P_i} - \frac{\partial f}{\partial P_i} \frac{\partial g}{\partial q_i} \right)$   
 $\frac{df}{dt} = \{f, H\} + \frac{\partial f}{\partial t}$  if conserved  $\{f, H\} = 0$   
 $\{q_i, q_j\} = 0 \quad \{q_i, P_j\} = \delta_{ij}$   
 $\{P_i, P_j\} = 0$

Central Force Motion

$F = F(r) \hat{r} \quad V = V(r) \quad \tau = r \times F = 0$   
 $L = \text{const.} \quad L = \frac{1}{2} m (\dot{r}^2 + r^2 \dot{\theta}^2) - V(r)$   
 $P_\theta = m r^2 \dot{\theta} = l \quad \dot{\theta} = \frac{l}{m r^2}$   
 $\frac{dA}{dt} = \frac{1}{2} r^2 \dot{\theta} = \frac{l}{2m}$   
 $E = \frac{1}{2} m \dot{r}^2 + \frac{1}{2} m r^2 \dot{\theta}^2 + V(r)$   
 $= \frac{1}{2} m \dot{r}^2 + V_{eff}(r)$   
 for  $V_{eff}(r) = V(r) + \frac{l^2}{2mr^2}$   
 allowed motion:  $E \geq V_{eff}(r)$   
 turning pts:  $V_{eff}(r) = E$   
 $m\ddot{r} = \frac{l^2}{mr^3} - \frac{dV}{dr} \quad m\ddot{r} = -\frac{dV_{eff}}{dr}$   
 $\frac{dV_{eff}}{dr} = 0 \equiv \frac{dV}{dr} = \frac{l^2}{mr^3}$   
 let  $u = \frac{1}{r}$ :  $\frac{du^2}{d\theta^2} + u = -\frac{m}{l^2 u^2} F(\frac{1}{u})$  for  $F(r) = -\frac{dV}{dr}$   
 $r(\theta) = \frac{p}{1 + e \cos \theta} \quad p = \frac{l^2}{mk} \quad e = \sqrt{1 + \frac{2El^2}{mk^2}}$

Systems of Particles  $M = \sum_i m_i$

$R = \frac{1}{M} \sum_i m_i r_i \quad P = M \dot{R} \quad \frac{dP}{dt} = F_{ext}^{tot}$   
 $M \ddot{R} = F_{ext}^{tot} \quad F_{ext}^{tot} = \sum_i F_i^{ext} \quad P_i = r_i - R$   
 $\sum_i m_i P_i = 0 \quad \sum_i m_i \dot{P}_i = 0 \quad T = T_{cm} + T_{rel}$   
 $= \frac{1}{2} M \dot{R}^2 + \frac{1}{2} \sum_i m_i \dot{P}_i^2$   
 $L = \sum_i r_i \times m_i \dot{r}_i = M R \times \dot{R} + \sum_i P_i \times m_i \dot{P}_i = L_{cm} + L_{rel}$   
 $\frac{dL}{dt} = \tau_{ext}^{tot} \quad V = \sum_i V_i^{ext}(r_i) + \sum_{i < j} V_{ij}(r_i - r_j)$   
 $= \frac{1}{2} \sum_{i < j} \sum_{j < i} V_{ij} \quad E = T + V$   
 for 2-body:  
 $\mu = \frac{m_1 m_2}{m_1 + m_2} \quad \mu \ddot{r} = -\nabla V(r)$  transl sym:  $P$  const  
 rot sym:  $L$  const  
 time transl sym:  $E$  const

Noninertial Frames

frame origin has accel  $A$   
 $F_{pseudo} = -mA \quad m a' = F - mA$  for rot w/ ang. vel  $\Omega$   
 for any vector  $Q \quad \left( \frac{dQ}{dt} \right)_{inertial} = \left( \frac{dQ}{dt} \right)_{rotating} + \Omega \times Q$   
 $v = v' + \Omega \times r \quad a = a' + 2\Omega \times v' + \Omega \times (r \times \Omega) + \ddot{r}$   
 if origin transl. w/ accel  $A$ :  $\ddot{r} = A$  to above  
 rot. frame eq. of motion  
 $m a' = F - mA - 2m\Omega \times v' - m\Omega \times (r \times \Omega) - m\ddot{r}$   
 Pseudo-forces:  $F_{tr} = -mA \quad F_{cor} = -2m\Omega \times v'$   
 $F_{cent} = -m\Omega \times (r \times \Omega) = m\Omega^2 r$  if  $r \perp \Omega$  w/ dist  $\rho$  from rot axis,  
 $F_{Euler} = -m \dot{\Omega} \times r \quad \left( \begin{matrix} F_{cent} = m\Omega^2 \rho \\ F_{Euler} = m\dot{\Omega} \times r \end{matrix} \right)$   
 $V_{eff} = V - \frac{1}{2} m |\Omega \times r|^2 = V - \frac{1}{2} m \Omega^2 \rho^2$   
 Equil  $e \frac{dV_{eff}}{dq} = 0$  stable if  $V''_{eff} > 0$

Rigid Bodies  $|r_i - r_j| = \text{const}$

$I = \sum_i m_i r_{i\perp}^2 \quad v_i = v_{cm} + \omega \times r_i$   
 $I = \int r_{\perp}^2 dm \quad I = I_{cm} + M d^2$   
 for planar bodies  
 $I_{ij} = \int (r^2 \delta_{ij} - x_i x_j) dm \quad I_z = I_x + I_y$  flat obj in xy plane  
 $L = I \omega \quad \tau = I \alpha$   
 Parallel axis thm. if  $I_{cm}$  is mom of inertia about a parallel axis displaced by  $d$  is  $I_{cm} + Md^2$ . rolling w/o slip:  $a = R\alpha$   
 $H = \frac{P_z^2}{2m} + \frac{P_\theta^2}{2m r^2} + V(r) \quad P_z = m \dot{z}$   
 $P_\theta = l = \text{const.}$   
 Near eq.  $r = r_0 + \delta$ ,  
 EOM:  $m \delta \ddot{r} + U'_{eff} \delta r = 0$   
 $e = \sqrt{1 + \frac{2El^2}{mk^2}}$

# Coupled Oscillators

$M\ddot{q} + Kq = 0$   $M$  is mass mat.  
 $K$  is stiffness mat.

Ansatz:  $q(t) = a \exp(i\omega t)$

$(K - \omega^2 M)a = 0$   $\det(K - \omega^2 M) = 0$

$(K - \omega^2 M)a = 0 \leftarrow$  mode shapes, solve for  $\omega$

$q(t) = \sum_n C_n a_n \cos(\omega_n t + \phi_n)$

$L = \frac{1}{2} \dot{q}^T M \dot{q} - \frac{1}{2} q^T K q$

E-L gives  $M\ddot{q} + Kq = 0$

Near eq  $K_{ij} = \frac{\partial^2 V}{\partial q_i \partial q_j}$

$V \approx V_0 + \frac{1}{2} \sum_{ij} K_{ij} \eta_i \eta_j$   $\eta_i = q_i - q_{eq}$

$a^T M a_s = 0$  for  $s \neq i$

normalized  $\equiv a^T M a_s = \delta_{rs}$

choose coords along normal modes

st.  $\ddot{Q}_r + \omega_r^2 Q_r = 0$

Beats: if 2 close freq are excited

$\omega_{beat} = |\omega_2 - \omega_1|$   $T_{beat} = \frac{2\pi}{|\omega_2 - \omega_1|}$

$T_{transfer} = \frac{T_{beat}}{2}$   $\omega^2 > 0$  stable  
 $\omega^2 = 0$  free mode  
 $\omega^2 < 0$  unstable

## More post-MT stuff

Rolling w/o slipping:  $\Delta x = R\Delta\theta$

$T = \frac{1}{2} I_{contact} \omega^2 = \frac{1}{2} (I_{CM} + Md_{contact,CM}^2) \omega^2$

If central force is inv-sq in distance, \*

$\frac{A}{r} = 1 + \epsilon \cos\theta$  where  $A = \frac{L^2}{mk}$  and

$\epsilon = \sqrt{1 + \frac{2EL^2}{mk^2}}$  is eccentricity.

ellipse when  $\epsilon < 1$  ( $\epsilon < 1$ )

parabola when  $\epsilon = 1$  ( $\epsilon = 1$ )

hyperbola when  $\epsilon > 1$  ( $\epsilon > 1$ )

## Coupled Oscillations

$L = T - V$   $T = \frac{1}{2} \dot{q}^T M \dot{q}$   $V = \frac{1}{2} q^T K q$

Elliptic Orbits  $\vec{F} = -\frac{k}{r^2} \hat{r}$

$r(\theta) = \frac{p}{1 + \epsilon \cos\theta}$   $p = \frac{L^2}{mk}$

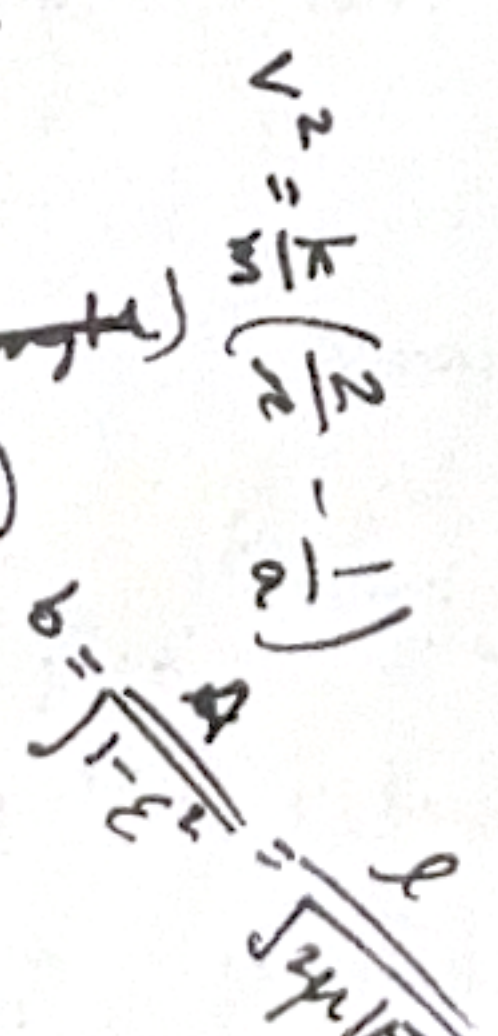
$a = -\frac{k}{2E}$  (semi-major axis)

$p = a(1 - \epsilon^2)$ ;  $l^2 = mka(1 - \epsilon^2)$

periapsis:  $r_{min} = a(1 - \epsilon) = \frac{p}{1 + \epsilon}$

apoapsis:  $r_{max} = a(1 + \epsilon) = \frac{p}{1 - \epsilon}$

Period:  $k = GMm$  (grav. orbit)  
 $T^2 = \frac{4\pi^2}{GM} a^3 \Rightarrow T = 2\pi \sqrt{\frac{ma^3}{k}}$



## Some Diff. Ansatzes

$\frac{dy}{dx} = f(x)g(y) \rightarrow \frac{dy}{g(y)} = dx/f(x)$

$y' + p(x)y = q(x) \rightarrow (y\mu)' = \mu q$   
 for  $\mu = \exp(\int p(x) dx)$

$y' = F(\frac{x}{z}) \rightarrow y = vx$

$ay'' + by' + cy = 0 \rightarrow y = e^{rx}$

for  $ra^2 + ba + c = 0$

## Poisson Brackets

$\frac{d}{dt} = \{f, H\} + \frac{\partial f}{\partial t}$   $\leftarrow$  no explicit time dep.

$\{g, f\} = -\{f, g\}$

if  $\{f, H\} = 0$ ,  $f$  is conserved.

$\{f, g\} = \sum_i \left( \frac{\partial f}{\partial q_i} \frac{\partial g}{\partial p_i} - \frac{\partial f}{\partial p_i} \frac{\partial g}{\partial q_i} \right)$

## Special Relativity

$S'$  moves sp. v. in +x dir rel to  $S$ .

$S \rightarrow S' : ct' = \gamma(ct - \beta x)$

$x' = \gamma(x - \beta ct)$

$y' = y$   $z' = z$

$\beta = \frac{v}{c}$ ,  $\gamma = \frac{1}{\sqrt{1 - \beta^2}}$

$\int \frac{dx}{x^2} = -\frac{1}{x} + C$

$\int \sin^2 x dx = \frac{x}{2} - \frac{\sin(2x)}{4}$

$\int \sin x dx = -\cos x + C$

$\int \frac{dx}{x^2} = -\frac{1}{x} + C$

$\int \frac{1}{1-x^2} dx = \frac{1}{2} \ln \left| \frac{1+x}{1-x} \right| + C$

$\frac{d}{dt}(a \cdot b) = \dot{a} \cdot b + a \cdot \dot{b}$

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about: intermediate is unstable

rotation about principal axis is stable

for torque-free rigid body

for rigid body in body-fixed

principal axes

for rigid body stuff

thin rod, end, perp to rod

thin rod, center, perp to rod

thin rod, center, perp to rod

thin rod, center, perp to rod

thin rod, center, perp to rod

$x = r \sin\theta \cos\phi$ ,  $y = r \sin\theta \sin\phi$

$r = \sqrt{x^2 + y^2 + z^2}$

$\theta = \tan^{-1} \left( \frac{\sqrt{x^2 + y^2}}{z} \right)$

$\phi = \tan^{-1} \left( \frac{y}{x} \right)$

$\frac{dV}{d\theta} = r^2 \sin\theta \frac{d\theta}{d\theta}$

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Elliptic  $I_a = \frac{1}{4} M(a^2 + b^2)$   
 $I_b = \frac{1}{4} M(a^2 + b^2)$   
 $I_c = M a^2 b^2$

Known moments of inertia  
 point mass  $I = m r^2$   
 thin rod, center, perp to rod  $I = \frac{1}{12} M L^2$   
 thin rod, end, perp to rod  $I = \frac{1}{3} M L^2$   
 solid disk, central axis  $I = \frac{1}{2} M R^2$   
 hoop, central axis  $I = M R^2$   
 solid sphere, through center  $I = \frac{2}{5} M R^2$   
 thin spherical shell, center  $I = \frac{2}{3} M R^2$   
 thin spherical shell, center  $I = \frac{2}{3} M R^2$

**Examples Block & Wedge**  
 no friction  
 wedge & block  
 block & floor  
 find  $a_M, N, a_m$  rel to M.  
 $N$  is norm force b/w  $m$  &  $M$ .  
 $M \ddot{x}_M = \frac{N \sin \theta}{M}$  In frame moving w/  $M$ , forces on  $m$  are  $mg, N$ , & inertial  $ma_M$   
 $N + ma_M \sin \theta - mg \cos \theta = 0$   
 $mg \sin \theta + ma_M \cos \theta = ma_M$   
 gives  $N = \frac{mMg \cos \theta}{M + m \sin^2 \theta}$   
 $a_M = \frac{mg \sin \theta \cos \theta}{M + m \sin^2 \theta}$   
 $\rightarrow mg \sin \theta + \frac{mgs \sin \theta \cos \theta}{M + m \sin^2 \theta}$   
 $\rightarrow a_m = g \sin \theta + \frac{mg \sin \theta \cos \theta}{M + m \sin^2 \theta}$   
 $= \frac{(m+M)g \sin \theta}{M + m \sin^2 \theta}$

**Rope on a Table**  
 rope of length  $l$  slides from frictionless tabletop. Released from rest w/ 30 cm hanging over edge of table. find time when left end of rope reaches end of table.  
 Tension does not do work to  $m$  since it's  $\perp$  to velocity.  
 $\frac{m \dot{l}^2}{2} \Rightarrow v$  const. Thus,  
 $wl = w(l_0 - a\theta) = w_0 l_0 - w_0 a \theta$   
 $w = w_0 l_0 / (l_0 - a\theta)$ .  $N_2 = \frac{dL_2}{dt}$   
 $N_2 = -T a$ .  $L_2 = mlv = mlw_0 l_0$   
 $= m w_0 l_0 (l_0 - a\theta)$   
 $\frac{dL_2}{dt} = -m w_0 l_0 a \dot{\theta} = \frac{-m a w_0^2 l_0^2}{(l_0 - a\theta)^2}$   
 $-T a = \frac{-m a w_0^2 l_0^2}{(l_0 - a\theta)^2}$  or  $T = \frac{m w_0^2 l_0^2}{(l_0 - a\theta)^2}$

**Spring w/ Friction**  
 Friction up to rel inst. vel.  
 $F_{fr} = -\beta(\dot{x}_1 - \dot{x}_2)$   
 $\begin{cases} m\ddot{x}_1 + \beta(\dot{x}_1 - \dot{x}_2) + kx_1 = 0 \\ m\ddot{x}_2 + \beta(\dot{x}_2 - \dot{x}_1) + kx_2 = 0 \end{cases}$   
 Ansatz:  $x_1(t) = B_1 e^{\alpha t}$   
 $x_2(t) = B_2 e^{\alpha t}$   
 where  $\alpha = 2 + i\omega$  is a quantity to be determined. Set det of coeffs of  $B_i$  to 0:  
 $(m\alpha^2 + \beta\alpha + k)^2 = \beta^2 \alpha^2$   
 $\rightarrow \alpha_1 = \pm i \sqrt{\frac{k}{m}}$ ;  $\omega_1 = \pm \sqrt{\frac{k}{m}}$   
 $\alpha_2 = \frac{1}{m}(-\beta \pm \sqrt{\beta^2 - mk})$

**Rigid Body Swinging about a Pivot**  
 Rigid body of mass  $M$  is pivoted about  $O$ . CoM is dist'd from pivot. Mom. of Inertia about pivot is  $I_0$ . Find small osc. freq.  
 $\tau = -Mgd \sin \theta$ ,  $I_0 \ddot{\theta} = -Mgd \sin \theta$   
 $\Rightarrow \omega = \sqrt{\frac{Mgd}{I_0}}$ .  $T = 2\pi \sqrt{\frac{I_0}{Mgd}}$

**Mass in a Slot on a Disk**  
 Disk rotating around vertical axis w/ ang. vel.  $\omega$ . Mass  $m$  moving in a slot w/o friction attached to 2 identical springs w/ eq. pos. in middle.  
 In rot frame w/ disk force acty on  $m$  is  $\vec{F} = -2ky\hat{e}_y + m\omega^2 x \hat{e}_x + \vec{N} - 2m\omega \hat{e}_z \times \dot{x} \hat{e}_x$   
 $= -2ky\hat{e}_y + m\omega^2 x \hat{e}_x + \vec{N} + 2m\omega \dot{x} \hat{e}_z$   
 $F_x = 0$ ,  $m\omega^2 x + N + 2m\omega \dot{x} = 0$   
 $F_y = m\ddot{y} - 2ky + m\omega^2 y = m\ddot{y}$   
 $\rightarrow \ddot{y} + (\frac{2k}{m} - \omega^2)y = 0$ . When  $\frac{2k}{m} > \omega^2$ ,  $y=0$  is a stable point. Osc. freq.  $\omega_0 = \sqrt{\frac{2k}{m} - \omega^2}$

**Bead on Rotating Hoop**  
 Bead of mass  $m$  slides w/o friction on circular hoop of radius  $R$ . lies in a vertical plane & rotates about vert. diam. w/ ang. vel.  $\Omega$ . Let  $\theta$  be angle of bead from lowest pt on hoop.  
 Find eq. pos. & determine when bottom pt is unstable.  
 $r_1 = R \sin \theta$ ,  $V_g = mgR(1 - \cos \theta)$   
 $V_c = -\frac{1}{2} m \Omega^2 r_1^2 = -\frac{1}{2} m \Omega^2 R^2 \sin^2 \theta$   
 $V_{eff} = V_g - V_c$ . Eq  $\frac{dV_{eff}}{d\theta} = 0$   
 $\rightarrow mR \sin \theta (g - \Omega^2 R \cos \theta) = 0$   
 $+ \sin \theta = 0$  or  $\cos \theta = \frac{g}{\Omega^2 R}$   
 stability near  $\theta = 0$ :  
 $V_{eff} = \frac{1}{2} mgR \theta^2 + \frac{1}{2} m \Omega^2 R^2 \theta^2$   
 stable if  $g - \Omega^2 R > 0$

**Rolling w/ Slipping**  $v_{cm} \neq \omega R$   
 object slides right w/ init speed  $v_0$ , no initial spin.  $a = -\mu_k g$   
 $\tau = f_k R = I \alpha$  & ass.  $I = kMR^2$   
 $\rightarrow a = \frac{\mu_k g}{kR}$   $v(t) = v_0 - \mu_k g t$   
 $\omega(t) = \frac{\mu_k g}{kR} t$   $v = \omega R$   
 $\rightarrow t = \frac{kv_0}{\mu_k g (k+1)}$

**Coriolis Force on Projectile**  
 projectile fired horizontally east @ latitude  $\lambda$ . Find direction & mag of Coriolis acceleration.  
 choose  $\hat{x}$  east,  $\hat{y}$  north,  $\hat{z}$  up  
 $\vec{a}_c = -2\vec{\Omega} \times \vec{v}$   
 at latitude  $\lambda$ ,  $\vec{z}$  up

**Center of Mass** two particles of mass  $m_1, m_2$ , pos given by  $\vec{r}_i$ .  
 Angular momentum wzt CoM is?  
 $\vec{R} = \frac{m_1 \vec{r}_1 + m_2 \vec{r}_2}{m_1 + m_2}$  so  $\vec{r}_{1c} = \vec{r}_1 - \vec{R}$   
 $\vec{r}_{2c} = \vec{r}_2 - \vec{R}$   
 $\vec{L} = \vec{r}_{1c} \times m_1 \dot{\vec{r}}_{1c} + \vec{r}_{2c} \times m_2 \dot{\vec{r}}_{2c}$   
 $= m_1 \left( \frac{m_2}{m_1 + m_2} \right) \vec{r}_1 \times \dot{\vec{r}}_1 + m_2 \left( \frac{m_1}{m_1 + m_2} \right) \vec{r}_2 \times \dot{\vec{r}}_2$

**Particle moving Radially on a Rotating Disk** Bead slides outward along frictional radial slot on horizontal disk rotating w/ ang. sp.  $\Omega$ . In rot. fr. of disk, bead moves outward w/ sp.  $\dot{x}$ . Find Coriolis force & sideways force exerted by slot on bead.  
 $\vec{v}_{rot} = \dot{x} \hat{r}$   
 $\Omega = \Omega \hat{z}$   
 $\vec{F}_{cor} = -2\Omega \times \vec{v}_{rot}$   
 $\hat{z} \times \hat{r} = \hat{\phi}$   
 $\rightarrow \vec{F}_{cor} = -2m\dot{x} \hat{\phi}$   
 Can't accel sideways rel. to slot  $\Rightarrow N_{slot} = -F_{cor}$

**Rolling w/o Slipping**  
 Solid cylinder of mass  $M$  & radius  $R$  rolls w/o slipping down incline of height  $h$ . Start from rest, find  $v$  at bottom.  
 $I = \frac{1}{2} MR^2$

**Cons. of Energy**  
 $Mgh = \frac{1}{2} Mv^2 + \frac{1}{2} I\omega^2$   
 $= \frac{1}{2} Mv^2 + \frac{1}{4} Mv^2$   
 $\rightarrow v = \sqrt{\frac{4gh}{3}}$

**Coupled Oscillators** with  $x_1(0) = A$ ,  $x_2(0) = 0$ ,  $\dot{x}_1(0) = \dot{x}_2(0) = 0$   
 EOM:  $m\ddot{x}_1 = -kx_1 - k_c(x_1 - x_2)$   
 $m\ddot{x}_2 = -kx_2 - k_c(x_2 - x_1)$   
 $w e q_+ = x_1 + x_2$   
 $q_- = x_1 - x_2$   
 $\rightarrow x_1 = \frac{q_+ + q_-}{2}$   
 $x_2 = \frac{q_+ - q_-}{2}$

**Work-Energy** Show W-E Thm is always correct in CM frame, i.e. if  $\vec{r}_{ic}, \vec{v}_{ic}$  are the pos & vel of  $m_i$  in CM frame.  
 $\sum_i \int \vec{F}_i \cdot d\vec{r}_{ic} = \Delta \sum_i \frac{m_i v_{ic}^2}{2}$

**Rope on a Table** (see above)  
 Let  $x$  hang over table &  $L$  tot length of rope.  $\frac{mgx}{L} = m \frac{dx^2}{dt^2}$   
 $\Rightarrow \ddot{x} = \frac{gx}{L}$ .  $x = Ae^{wt} + Be^{-wt}$   
 $\rightarrow \omega = \sqrt{\frac{2g}{L}}$ . from init. cond.  $A=B = \frac{x_0}{2}$ ,  $x = x_0 \cosh(\omega t)$

**Falling Cube** edge length  $l$  (homogeneous) has one edge on a plane. Find angular vel when face strikes plane.  
 (a) No sliding. Energy conv.  $mg \frac{l}{\sqrt{2}} = mg \frac{l}{2} + \frac{mv_{cm}^2}{2} + \frac{I\omega^2}{2}$  for  $v_{cm}$  is vel of CM when one face strikes plane.  $v_{cm} = \frac{l}{\sqrt{2}} \omega$ .  $I = \frac{1}{6} ml^2$   
 $\rightarrow \frac{mg l}{2} (\sqrt{2} - 1) = \frac{1}{3} ml^2 \omega^2$   
 $\rightarrow \omega^2 = \frac{3}{2} \frac{g}{l} (\sqrt{2} - 1)$

adding eqns:  
 $m\ddot{q}_+ = -kq_+$   
 $\rightarrow \omega_+ = \sqrt{\frac{k}{m}}$   
 subtracting eqns:  
 $m\ddot{q}_- = -(k + 2k_c)q_-$   
 $\rightarrow \omega_- = \sqrt{\frac{k + 2k_c}{m}}$   
 from init. cond.  
 $q_+(t) = A \cos(\omega_+ t)$   
 $q_-(t) = A \cos(\omega_- t)$

In the inertial frame  $\sum_i \int \vec{F}_i \cdot d\vec{r}_i = \Delta \sum_i \frac{m_i v_i^2}{2}$   
 Using  $\vec{r}_i = \vec{r}_{ic} + \vec{r}_c$   
 $\sum_i \int \vec{F}_i \cdot d\vec{r}_{ic} + \sum_i \int \vec{F}_i \cdot d\vec{r}_c = \Delta \sum_i \frac{m_i}{2} (v_{ic}^2 + v_c^2 + 2\vec{v}_{ic} \cdot \vec{v}_c)$   
 $= \Delta \sum_i \frac{m_i}{2} v_{ic}^2 + \Delta \sum_i \frac{m_i}{2} v_c^2 + \Delta \sum_i m_i \vec{v}_{ic} \cdot \vec{v}_c$ .  $\sum_i m_i \vec{v}_{ic} = 0$   
 $\rightarrow \sum_i \int \vec{F}_i \cdot d\vec{r}_{ic} + \int \vec{F}_{tot} \cdot d\vec{r}_c = \Delta \sum_i \frac{m_i}{2} v_{ic}^2 + \Delta \frac{M v_c^2}{2}$   
 $\vec{F}_{tot} = \sum_i \vec{F}_i = \sum_i m_i \ddot{\vec{r}}_i = M \ddot{\vec{r}}_c \Rightarrow \int \vec{F}_{tot} \cdot d\vec{r}_c = M \int \frac{d\vec{v}_c}{dt} \cdot \vec{v}_c = \Delta \frac{M v_c^2}{2}$   
 subtract from  $\Delta \frac{M v_c^2}{2}$  and conclude.

(b) Sliding.  $\theta$  angle from vertical (wrt  $0, \frac{\pi}{4}$ ).  
 $y = \frac{l}{\sqrt{2}} \cos \theta$   $\dot{y} = -\frac{l}{\sqrt{2}} \sin \theta \dot{\theta}$   
 $= \frac{1}{2} l \dot{\theta}^2 = -\frac{1}{2} l \omega^2 \sin^2 \theta$   
 cons. of energy gives  $\omega^2 = \frac{11}{5} \frac{g}{l} (\sqrt{2} - 1)$

convert + trig. ids.  
 $x_1(t) = A \cos(\frac{\omega_+ + \omega_-}{2} t) \cos(\frac{\omega_- - \omega_+}{2} t)$   
 $x_2(t) = A \sin(\frac{\omega_+ + \omega_-}{2} t) \sin(\frac{\omega_- - \omega_+}{2} t)$   
 $A_1(t) = A \left| \cos(\frac{\omega_- - \omega_+}{2} t) \right|$   
 $A_2(t) = A \left| \sin(\frac{\omega_- - \omega_+}{2} t) \right|$   
 transfer when  $A_1 = 0, A_2 = A$   
 $t_{transfer} = \frac{\pi}{\omega_- - \omega_+}$   $t_{ret} = 2 t_{transfer}$